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Iterative Coupling of Boundary and Discrete Element Method Using an Overlapping FEM Zone

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- 2 Numerical Methods
- 3 Inelastic zone detection
- 4 Iterative Coupling







2 Numerical Methods

3 Inelastic zone detection



Motivation





- Inelastic zones not known a priori
- May occurs in small portions of total domain
- Not all numerical methods suitable for modelling



Develop a Methodology

Linear elastic analysis

Detect zones with inelastic material behaviour

Define new regions and remesh

Solve the coupled problem

Involve different methods where they work best

- o Boundary Element Method
- o Discrete Element Method



2 Numerical Methods

3 Inelastic zone detection





- Continuum theory
- Two basic fields:
 - o Displacements: $\mathbf{u} = (u_x, u_y, u_z)$
 - Tractions: $\mathbf{t} = (t_x, t_y, t_z) = \boldsymbol{\sigma} \cdot \hat{\mathbf{n}}$

•
$$\mathbf{c} \cdot \mathbf{u} = \int_{S} \mathbf{U}^* \cdot \mathbf{t} \, dS - \int_{S} \mathbf{T}^* \cdot \mathbf{u} \, dS$$

Stresses on the boundary

- Material represented as collection of particles
- Particles interacts with contact forces
- Dynamic method Newton's Second Law







Equation of motion

$$o \sum_{i=1}^{N} \left[\left(m_{i} \ddot{\mathbf{u}}_{i} - \mathbf{F}_{i} \right) \delta u_{i} + \left(\mathbf{I}_{i} \dot{\boldsymbol{\omega}}_{i} - \mathbf{T}_{i} \right) \delta \phi_{i} \right] = 0$$

o Motion equations integrated in time with central difference scheme

Fields:

o Displacements: $\mathbf{u}_{i} = (u_{x}, u_{y}, u_{z})$ o Forces: $\mathbf{F}_{i} = (F_{x}, F_{y}, F_{z}) = F_{i}^{ext} + \sum_{j=1}^{n_{i}^{cont}} F_{ij}^{cont} + F_{i}^{damp}$

• Angular velocities: $\omega_i = (\omega_x, \omega_y, \omega_z)$

o Angular momentum: $\mathbf{T}_{i} = (T_{x}, T_{y}, T_{z}) = T_{i}^{ext} + \sum_{j=1}^{n_{i}^{cont}} S_{ij}^{c} \times F_{ij}^{cont} + T_{i}^{damp}$



2 Numerical Methods

3 Inelastic zone detection





Assumption:

Zone starts from the boundary

- Make an elastic analysis
- Criteria expressed as a yield surface $F(\sigma) = \sigma_{max} T$
- Follow the isocurve

























2 Numerical Methods

3 Inelastic zone detection





- No global stiffness matrix available
- Static BEM dynamic DEM: a quasi-static approach
- Each method retains its advantages
- Different programming languages
- Add a FEM overlapping zone for smooth transition between DEM and BEM



FEM formulation

From Hamilton's principle

$$\mathbf{M}_F \ddot{\mathbf{u}}_F = \mathbf{F}_F^{ext} - \mathbf{F}_F^{int}$$

Assembled from elemental matrices and vectors:

$$\begin{split} \mathbf{m}_{e} &= \int_{\Omega_{e}} \rho \mathbf{N}^{T} \mathbf{N} d\Omega_{e} \\ \mathbf{f}_{e}^{int} &= \int_{\Omega_{e}} \mathbf{B}^{T} \boldsymbol{\sigma} d\Omega_{e} \\ \mathbf{f}_{e}^{ext} &= \int_{\Omega_{e}} \mathbf{N}^{T} \rho \mathbf{b} d\Omega_{e} + \int_{\Gamma_{e}} \mathbf{N}^{T} \mathbf{t} d\Gamma_{e} \\ &\text{Integrated in time with central difference scheme} \end{split}$$



Transition Zone

- Total energy as a linear combination $\partial W = \alpha \partial W_F + (1 - \alpha) \partial W_D$
- Kinematic constraints:

 $\mathbf{u}_{d_i} = \mathbf{N} \mathbf{r}_F^{\mathrm{e}}$

 $\dot{\mathbf{u}}_{d_i} = \mathbf{N}\dot{\mathbf{r}}_F^e$

 $\ddot{\mathbf{u}}_{d_i} = \mathbf{N} \ddot{\mathbf{r}}_F^e$

 Penalty method for imposing the above constraints



Iterative Coupling





• sub-domain (1):

$$\mathbf{u}_1 = \{\mathbf{u}_1^1, \mathbf{u}_1'\}^T$$

 $\mathbf{f}_1 = \{\mathbf{f}_1^1, \mathbf{f}_1'\}^T$
• sub-domain (2):
 $\mathbf{u}_2 = \{\mathbf{u}_2^2, \mathbf{u}_2'\}^T$
 $\mathbf{t}_2 = \{\mathbf{t}_2^2, \mathbf{t}_2'\}^T$

Compatibility and equilibrium conditions

•
$$\mathbf{u}_1' = \mathbf{u}_2'$$

• $\mathbf{f}_1' = -\mathbf{f}_2'$

Iterative Coupling





• sub-domain (1):

$$\mathbf{u}_1 = {\mathbf{u}_1^1, \mathbf{u}_1'}^T$$

 $\mathbf{f}_1 = {\mathbf{f}_1^1, \mathbf{f}_1'}^T$
• sub-domain (2):
 $\mathbf{u}_2 = {\mathbf{u}_2^2, \mathbf{u}_2'}^T$
 $\mathbf{t}_2 = {\mathbf{t}_2^2, \mathbf{t}_2'}^T$

Compatibility and equilibrium conditions

•
$$u'_1 = u'_2$$

• $f'_1 = -f'_2$



BEM – DEM/FEM Region Coupling

- Methods with different fields
- Nodal point forces to be calculated on the BEM side on the interface

BEM Region: Calculate stiffness matrix for the interface

 $\mathbf{o} \ \left\{\mathbf{t}\right\}_{c} = \left\{\mathbf{t}\right\}_{c0} + \mathbf{K}_{BE} \left\{\mathbf{u}\right\}_{c} \quad \Rightarrow \quad \left\{\mathbf{F}\right\}_{c} = \left\{\mathbf{F}\right\}_{c0} + \mathbf{K} \left\{\mathbf{u}\right\}_{c}$

Relaxation



- Exchange displacements and forces an the interface
- Iterative procedure requires relaxation to converge
- Applied BC is a linear combination
 - o Previous step solution
 - Boundary condition for n + 1 iteration step provided from adjacent region

$$\mathbf{u}_{n+1}^{1} = \alpha \mathbf{u}_{n}^{1} + (1 - \alpha) \mathbf{u}_{n+1}^{2}$$
$$\mathbf{f}_{n+1}^{1} = \beta \mathbf{f}_{n}^{1} + (1 - \beta) \mathbf{f}_{n+1}^{2}$$

 \blacksquare Bad news: Relaxation parameters $\alpha,\,\beta$ are problem dependent

Good news: Relaxation can be automated



- We need to know when to stop
- Displacement based criterion
- User sets maximum relative error between two sequential steps

• Criterion used:
$$\frac{||\mathbf{u}_{n+1}' - \mathbf{u}_n'||}{||\mathbf{u}_{n+1}'||} < \epsilon$$







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DEM - BEM results















Thank You!

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